

ENDURANCE LIMIT MODIFYING FACTORS

FOR A REAL PART, THE MEASURED ENDURANCE LIMIT MUST BE MODIFIED TO REFLECT A REAL PART OPERATING IN A REAL ENVIRONMENT. THIS CAN BE ACCOMPLISHED THROUGH MARIN'S EQUATION, WHICH IS WRITTEN AS

$$S_e = k_a k_b k_c k_d k_e S'_e$$

WHERE k_a = SURFACE FACTOR

k_b = SIZE FACTOR

k_c = LOAD FACTOR

k_d = TEMPERATURE FACTOR

k_e = RELIABILITY FACTOR

MARIN FACTORS

S'_e = ROTARY-BEAM TEST SPECIMEN ENDURANCE LIMIT

S_e = ENDURANCE LIMIT AT THE CRITICAL LOCATION OF A MACHINE PART IN THE GEOMETRY AND CONDITION OF USE

SURFACE FACTOR, k_a

ACCOUNTS FOR INCREASED SUSCEPTIBILITY TO FATIGUE WHEN SURFACE FINISHES ARE ROUGH.

$$k_a = a S_{ut}^b$$

WHERE a AND b ARE CURVE FIT PARAMETERS THAT DEPEND ON YOUR SURFACE FINISH (i.e. GROUND, MACHINED, HOT-ROLLED, AS-FORGED)

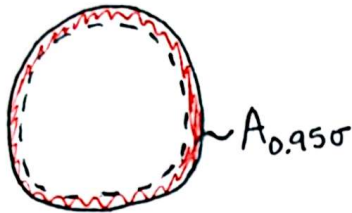
SIZE FACTOR, k_b

ACCOUNTS FOR LARGER NUMBER OF POTENTIAL FATIGUE INITIATION SITES AS PARTS INCREASE IN SIZE.

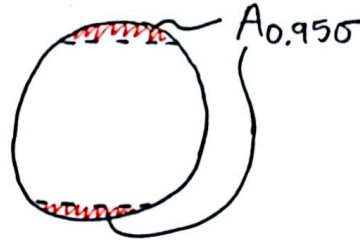
FOR BENDING OR TORSION	$k_b = \begin{cases} (d/0.3)^{-0.107} = 0.879d^{-0.107} & , 0.3 \leq d \leq 2 \text{ in} \\ 0.91d^{-0.157} & , 2 < d \leq 10 \text{ in} \\ (d/7.62)^{-0.107} = 1.24d^{-0.107} & , 7.62 \leq d \leq 51 \text{ mm} \\ 1.51d^{-0.157} & , 51 < d \leq 254 \text{ mm} \end{cases}$	FOR AXIAL, $k_b = 1$
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THESE EQUATIONS FOR k_b ASSUME A ROUND ROTATING SPECIMEN IN BENDING OR TORSION. FOR OTHER SCENARIOS, WE NEED TO FIND AN EQUIVALENT DIAMETER TO USE. THE EQUIVALENT DIAMETER, d_e IS FOUND BY EQUATING THE VOLUME OF MATERIAL STRESSED AT AND ABOVE 95% OF THE MAXIMUM STRESS TO THE SAME VOLUME IN THE ROTATING-BEAM SPECIMEN.

ROTATING-BEAM SPECIMEN



NON-ROTATING BENDING



SEE TABLE 6-3 IN SHIGLEY FOR EFFECTIVE DIAMETERS OF VARIOUS GEOMETRIES

LOAD FACTOR, k_c

ACCOUNTS FOR DIFFERENT STRESS DISTRIBUTIONS IN BENDING VS. AXIAL VS. TORSIONAL LOADING, AND DIFFERENT STRENGTH VALUES IN SHEAR VS. NORMAL.

$$k_c = \begin{cases} 1 & \text{BENDING} \\ 0.85 & \text{AXIAL} \\ 0.59 & \text{TORSION} \end{cases}$$

TEMPERATURE FACTOR, k_d

ACCOUNTS FOR CHANGES IN STRENGTH AND DUCTILITY WITH CHANGES IN TEMPERATURE.

- IF YOU HAVE S_e' AT THE OPERATING TEMPERATURE FOR YOUR PART, USE THAT AND LET $k_d = 1$.
- IF YOU HAVE S_e' AT ROOM TEMPERATURE AND NEED TO ADJUST FOR THE OPERATING TEMPERATURE, USE

$$k_d = S_T / S_{RT}$$

WHERE S_T AND S_{RT} ARE THE ULTIMATE STRENGTHS AT OPERATING TEMPERATURE AND ROOM TEMPERATURE, RESPECTIVELY.

FOR STEELS,

$$S_T / S_{RT} = 0.98 + 0.35(10^{-4})T_F - 6.3(10^{-7})T_F^2 \quad 70^\circ\text{F} \leq T_F \leq 1000^\circ\text{F}$$

$$S_T / S_{RT} = 0.99 + 0.59(10^{-4})T_C - 2.1(10^{-6})T_C^2 \quad 20^\circ\text{C} \leq T_C \leq 550^\circ\text{C}$$

- IF YOU ONLY HAVE S_{RT} , USE THE ABOVE TO ESTIMATE S_T AND THEN LET $S_p' = 0.5 S_T$ (FOR $S_T \leq 200$ ksi) WITH $k_d = 1$.

RELIABILITY FACTOR, k_e

ADJUSTS ENDURANCE LIMIT ESTIMATE TO CAUSE FEWER DATA POINTS TO LIE BELOW THE ESTIMATED VALUE.

$$k_e = 1 - 0.08 Z_\alpha$$

WHERE Z_α IS TRANSFORMATION VARIATE FOR A NORMAL DISTRIBUTION.

$$Z_\alpha = \frac{X - \mu_x}{\hat{\sigma}_x}$$

AND VALUES FOR ANY DESIRED RELIABILITY CAN BE DETERMINED FROM TABLES IN SHIGLEY (TABLE A-20 AND/OR TABLE 6-4).

MISCELLANEOUS EFFECTS

ANY NUMBER OF ADDITIONAL FACTORS COULD BE INCLUDED AS ADDITIONAL MULTIPLIERS IN THE MARIN EQUATION (k_f, k_g , etc.). A FEW EFFECTS THAT MAY WARRANT AN ADDITIONAL MARIN FACTOR ARE LISTED BELOW:

- RESIDUAL STRESSES
- DIRECTIONAL CHARACTERISTICS
- CORROSION
- ELECTROLYTIC PLATING
- METAL SPRAYING
- CYCLIC FREQUENCY
- FRETAGE CORROSION